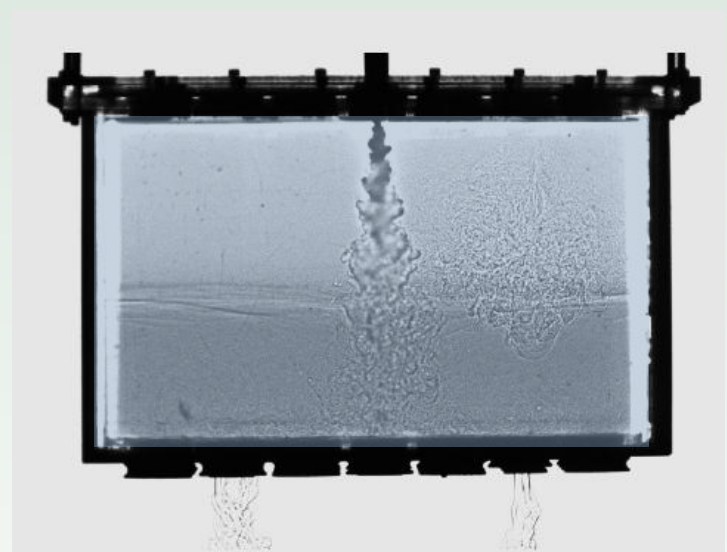
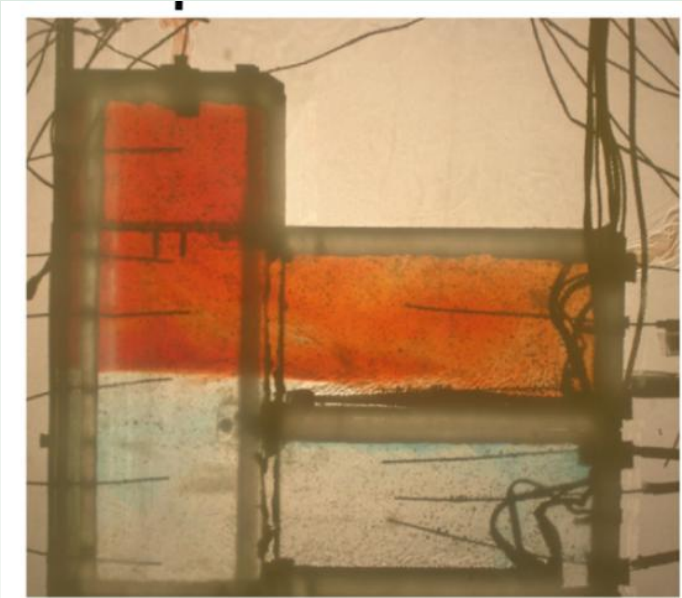


# Modelling natural ventilation: Small-scale experiments in water

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Andy Acred



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**UNIVERSITY OF  
CAMBRIDGE**

## Introduction to water bath modelling

- Why experiment?
- Overview of techniques: heat & salt

## Capabilities: some case studies

- Combined wind and buoyancy
- Discharge coefficient and exchange flows

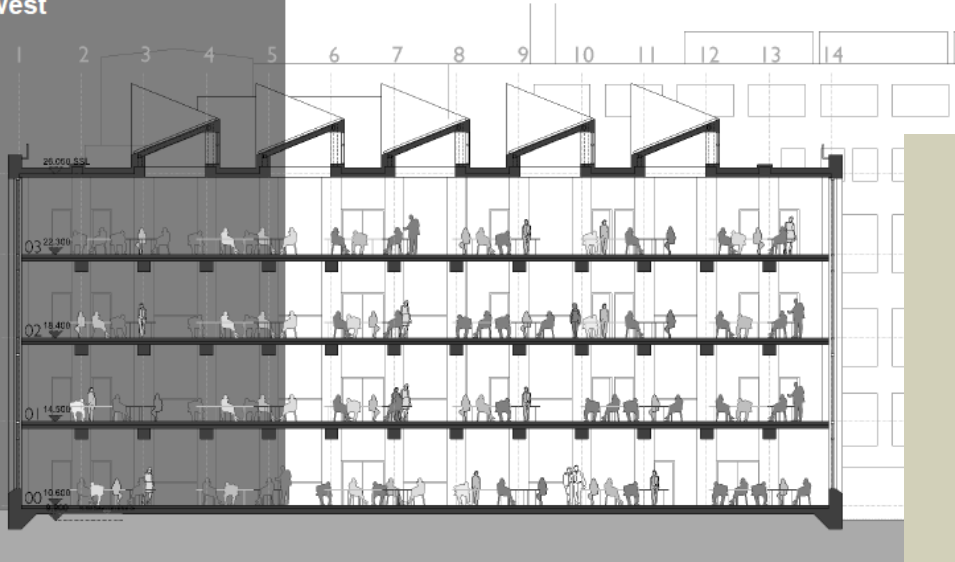
# Why experiment?



- Real-time, 3-D fluid flows
- Flow visualisation
- Capture specific flow phenomena
- Validate theory
- Push forward intuitive understanding

# The experimental method

Section  
looking west



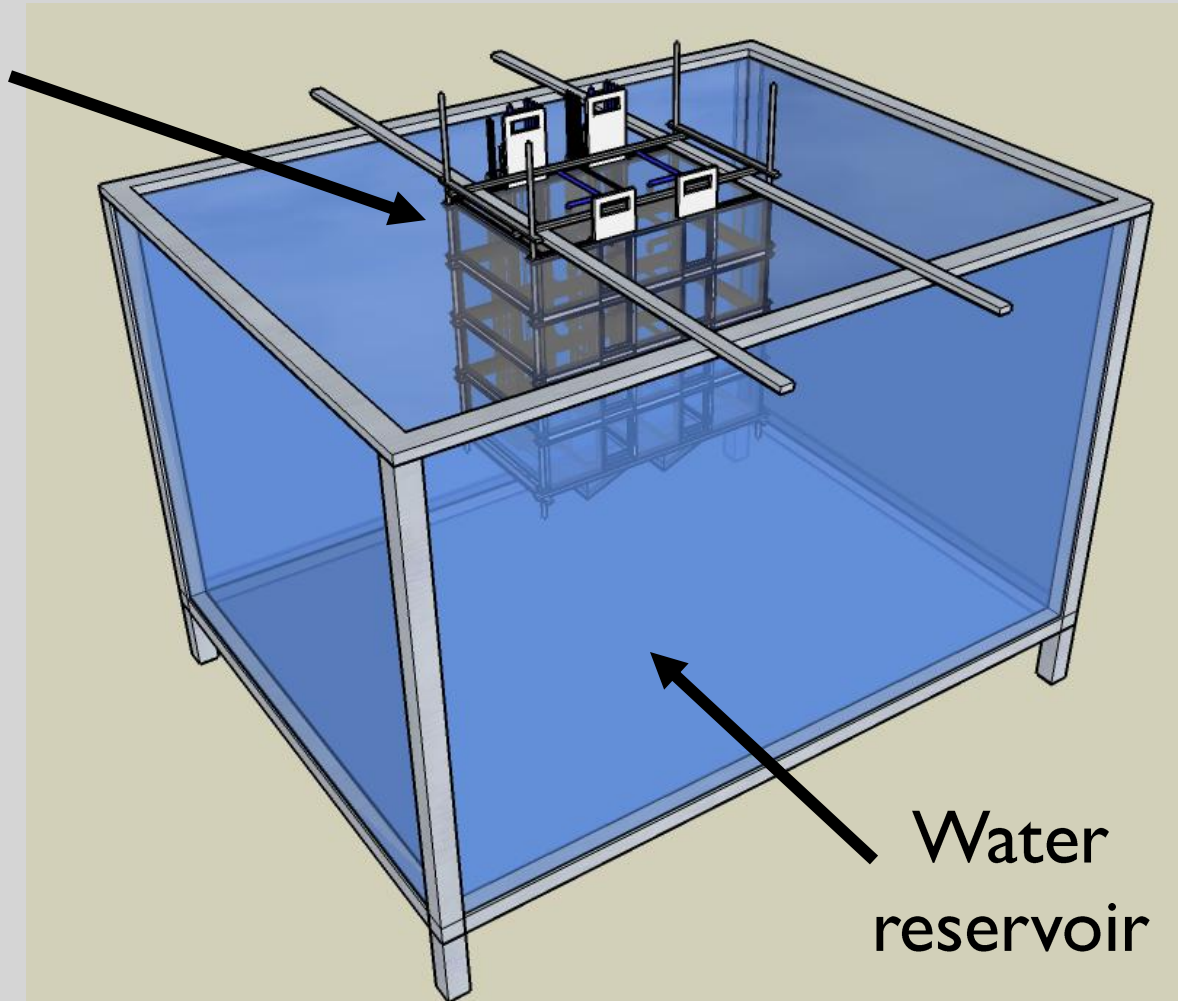
Full-scale design

1:20 – 1:50  
scale model



# The experimental method

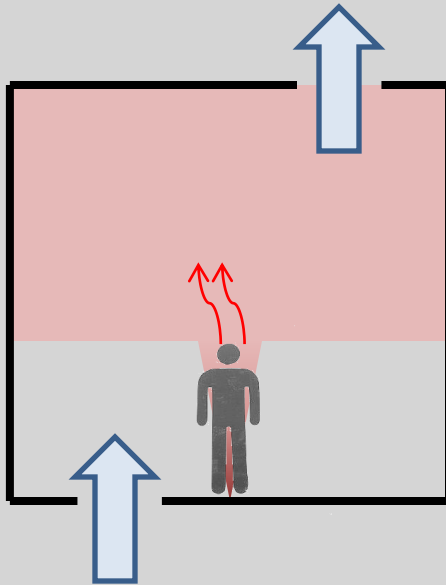
Building  
model



# Buoyancy sources

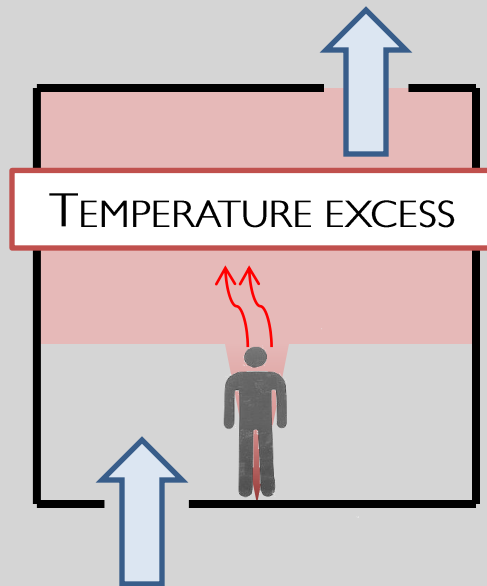
FULL SCALE

Heat in air



FULL SCALE

Heat in air

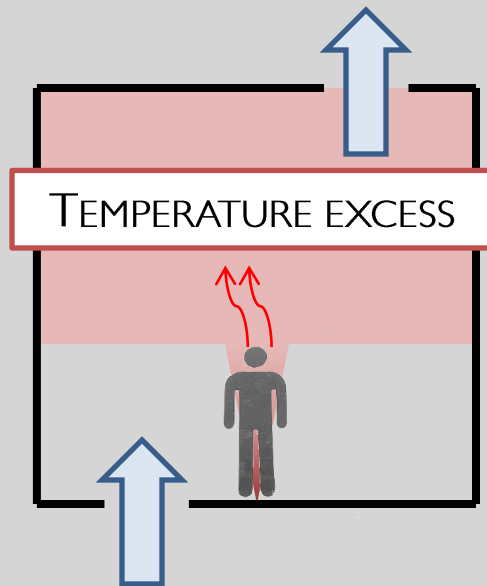


# Buoyancy sources



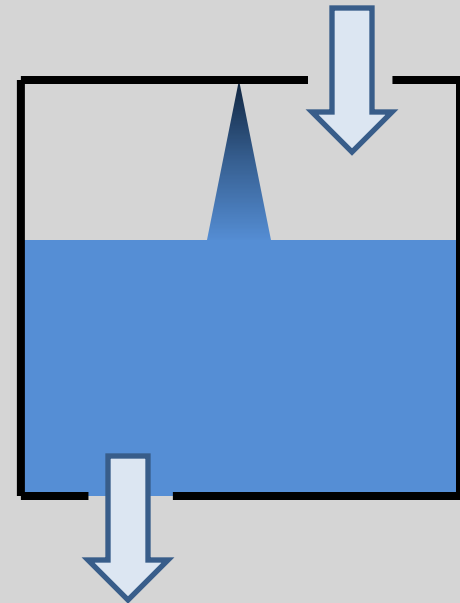
FULL SCALE

Heat in air



MODEL SCALE

Saline in water

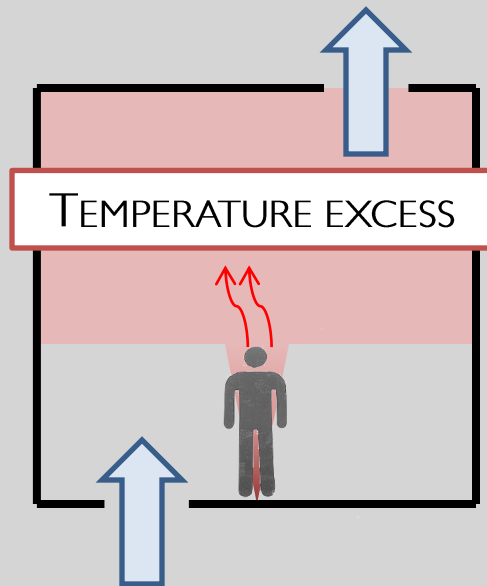




# Buoyancy sources

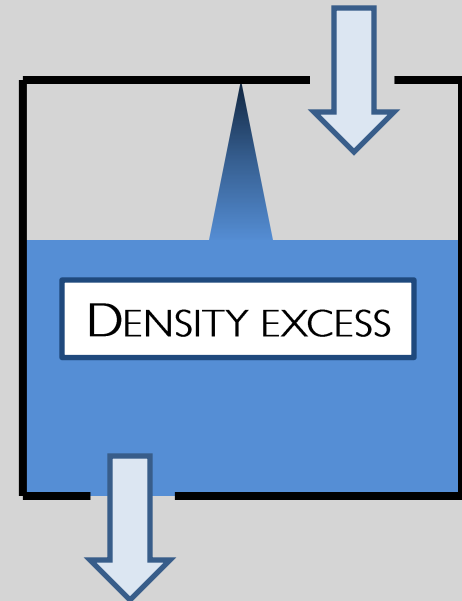
FULL SCALE

Heat in air



MODEL SCALE

Saline in water



# Buoyancy sources

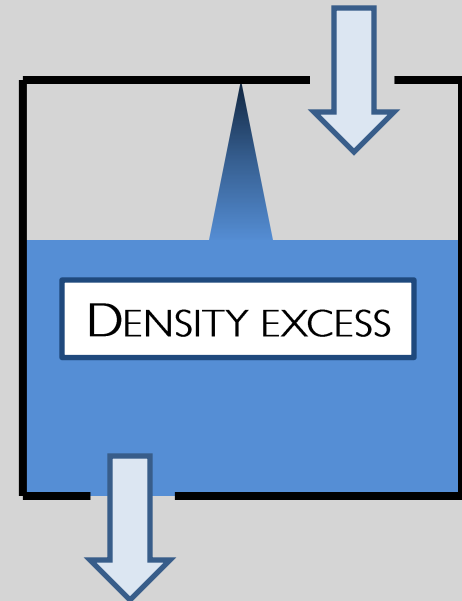
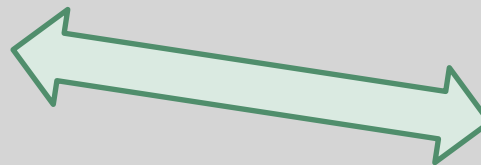
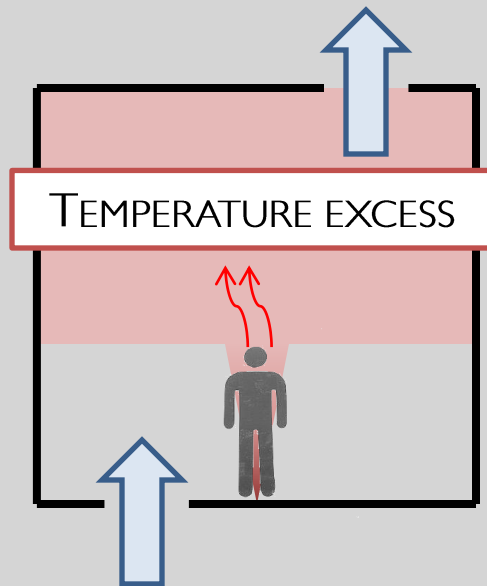
FULL SCALE

EQUIVALENT  
FLUID DYNAMICS

MODEL SCALE

Heat in air

Saline in water



# Buoyancy sources

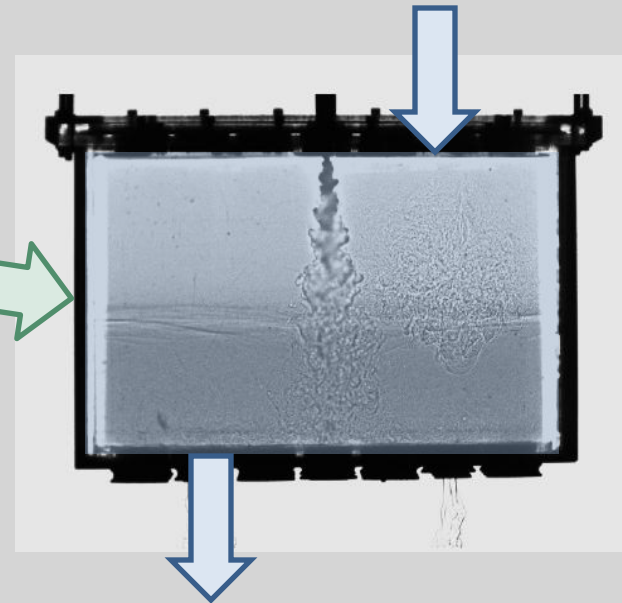
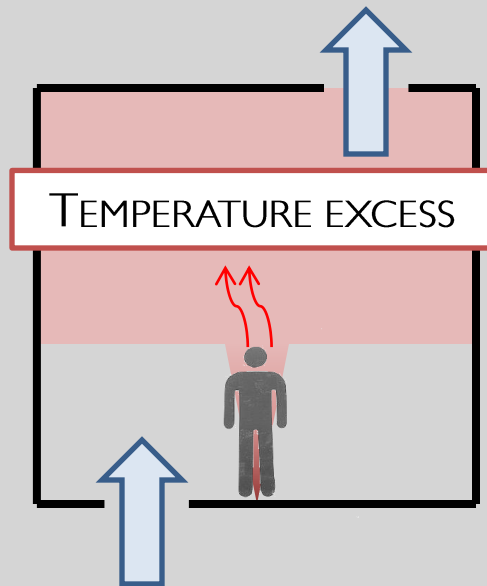
FULL SCALE

Heat in air

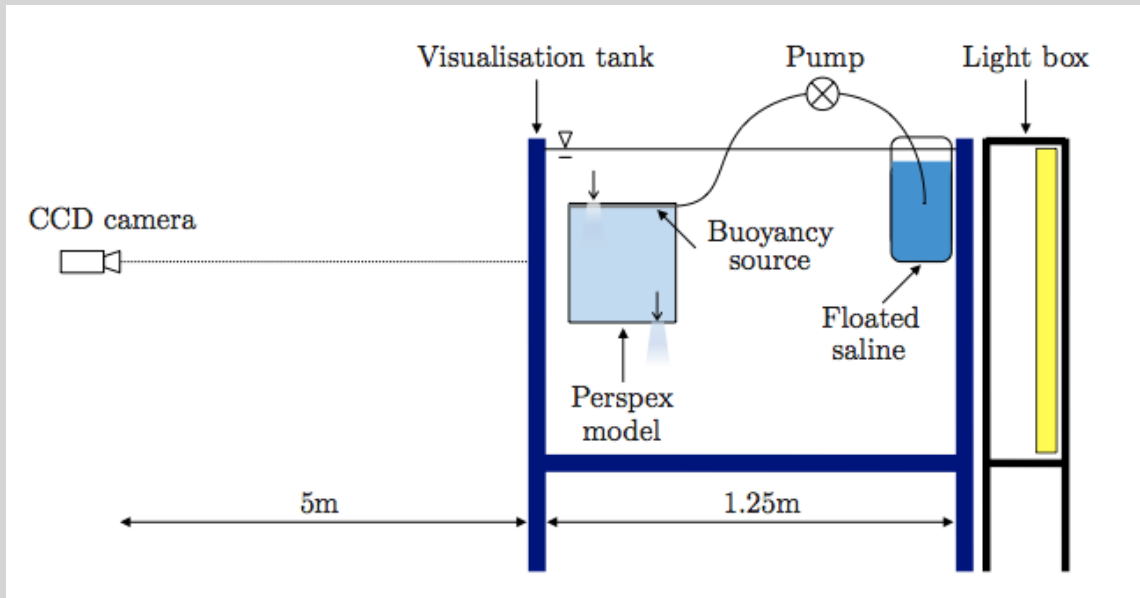
EQUIVALENT  
FLUID DYNAMICS

MODEL SCALE

Saline in water



# Dye attenuation technique



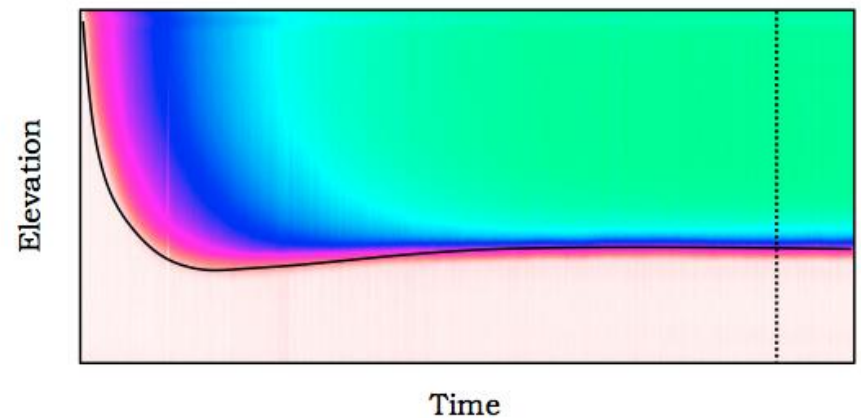
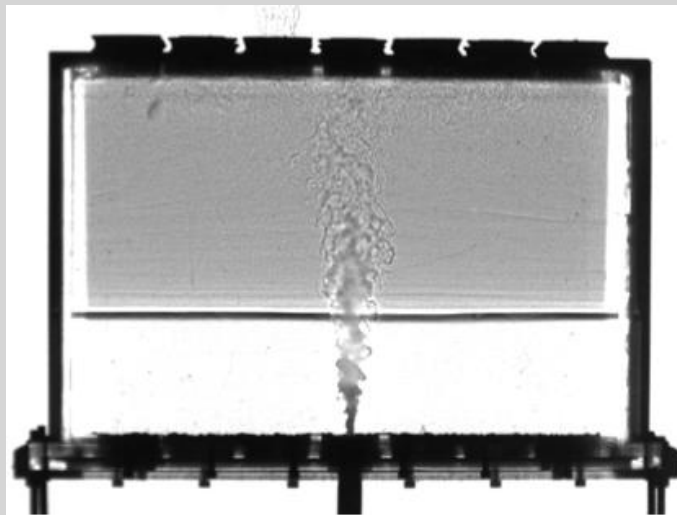
Light transmittance



Salinity



Temperature  
at full scale



# Buoyancy sources

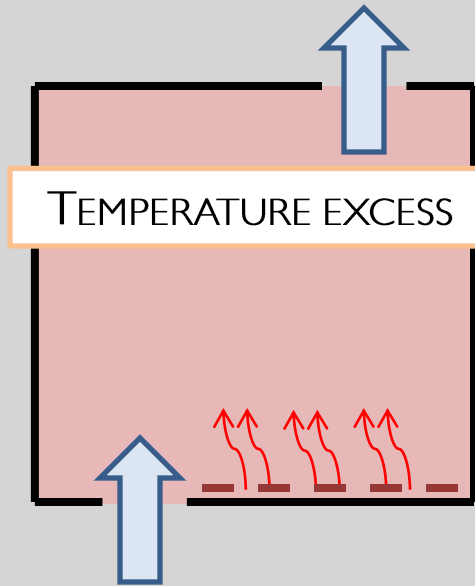
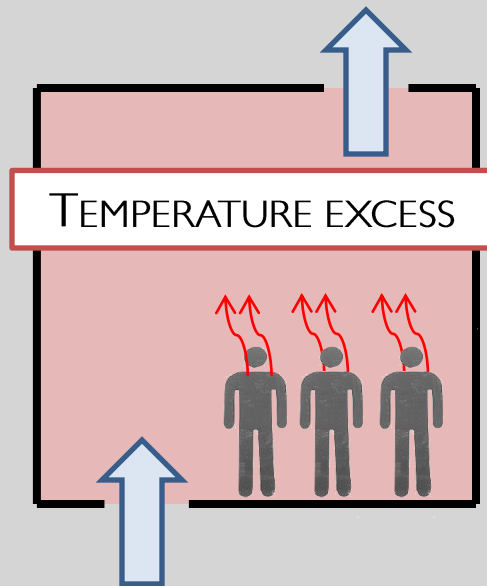
FULL SCALE

EQUIVALENT  
FLUID DYNAMICS

MODEL SCALE

Heat in air

Heat in water



FULL SCALE

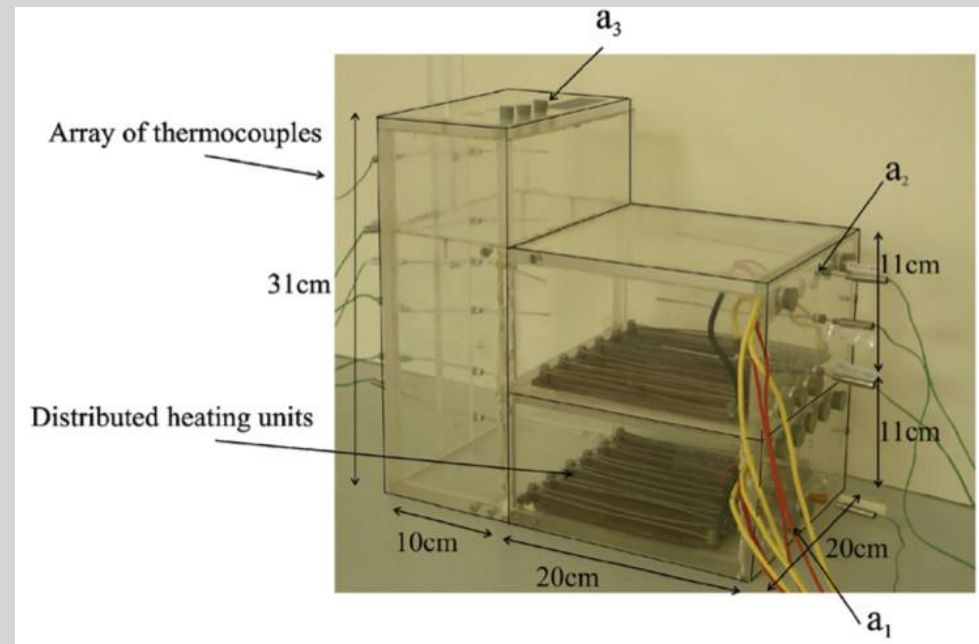
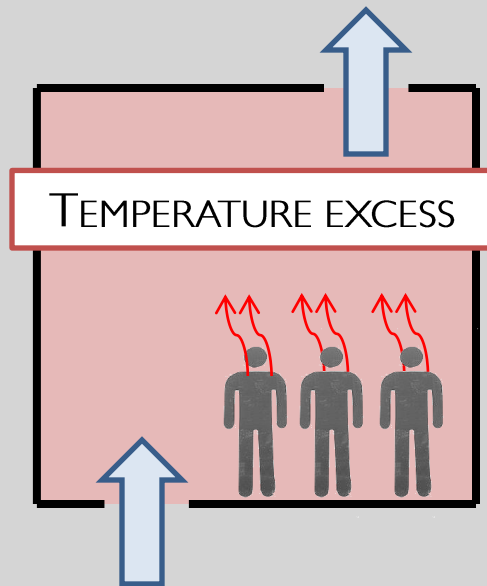
EQUIVALENT  
FLUID DYNAMICS

MODEL SCALE

Heat in air



Heat in water



# Buoyancy sources

FULL SCALE

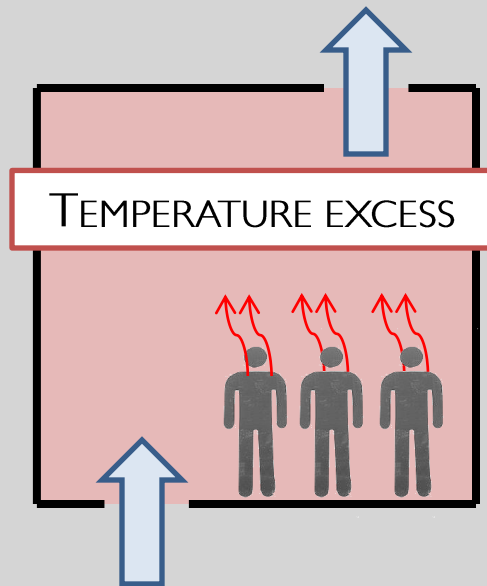
Heat in air

EQUIVALENT  
FLUID DYNAMICS



MODEL SCALE

Heat in water



# Dynamic similarity

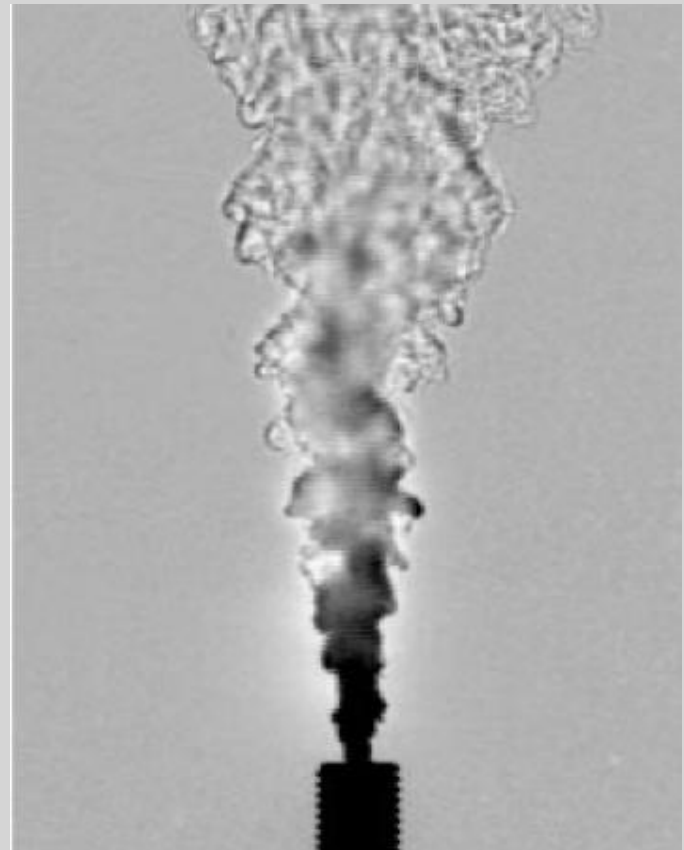
$$Re = \frac{\text{Inertia}}{\text{Viscosity}} \gtrsim 10^3$$

$$Pe = \frac{\text{Advection}}{\text{Diffusion}} \gtrsim 10^3$$

$$Ra = \frac{\text{Convection}}{\text{Conduction}} \gtrsim 10^8$$

$$Fr = \frac{\text{Inertia}}{\text{Buoyancy}}$$

Turbulent flows in which heat (buoyancy) transport occurs by advection





# Dynamic similarity

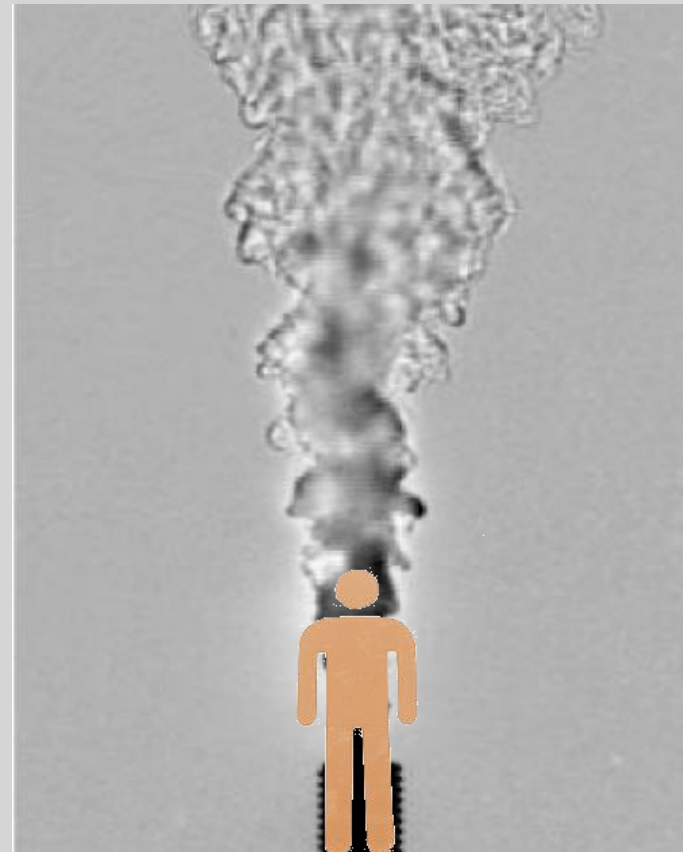
$$Re = \frac{\text{Inertia}}{\text{Viscosity}} \gtrsim 10^3$$

$$Pe = \frac{\text{Advection}}{\text{Diffusion}} \gtrsim 10^3$$

$$Ra = \frac{\text{Convection}}{\text{Conduction}} \gtrsim 10^8$$

$$Fr = \frac{\text{Inertia}}{\text{Buoyancy}}$$

Turbulent flows in which heat (buoyancy) transport occurs by advection



# Dynamic similarity

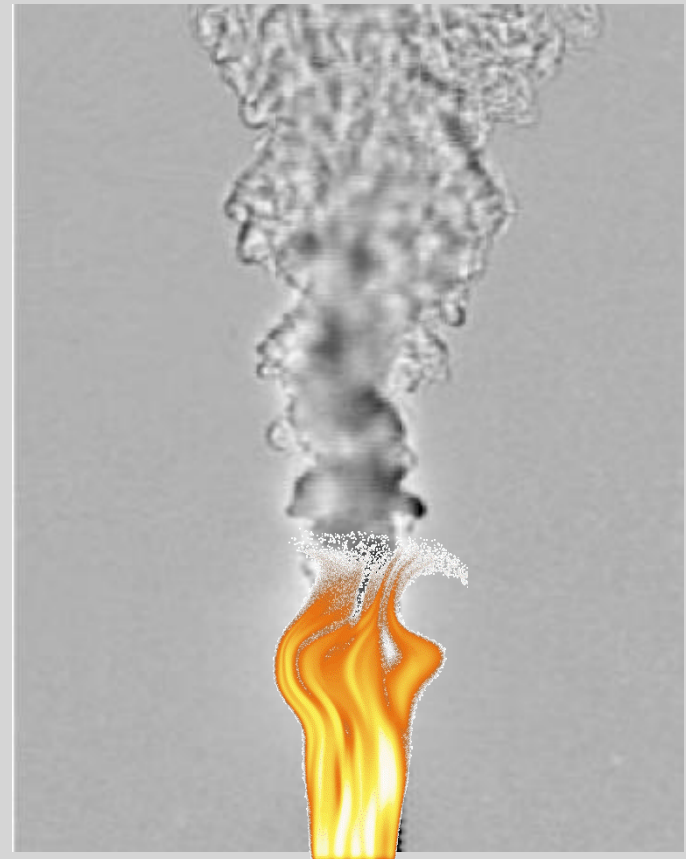
$$Re = \frac{\text{Inertia}}{\text{Viscosity}} \gtrsim 10^3$$

$$Pe = \frac{\text{Advection}}{\text{Diffusion}} \gtrsim 10^3$$

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Turbulent flows in which heat (buoyancy) transport occurs by advection

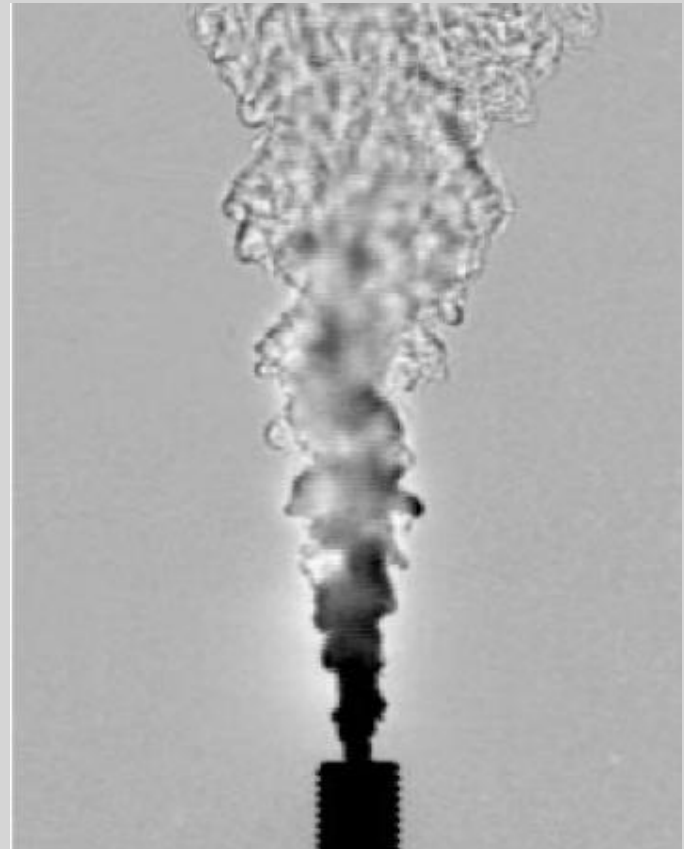


## Heat in water



PARTRIDGE & LINDEN (2013)

## Salt in water



HUNT & LINDEN (2001)

## Heat in water

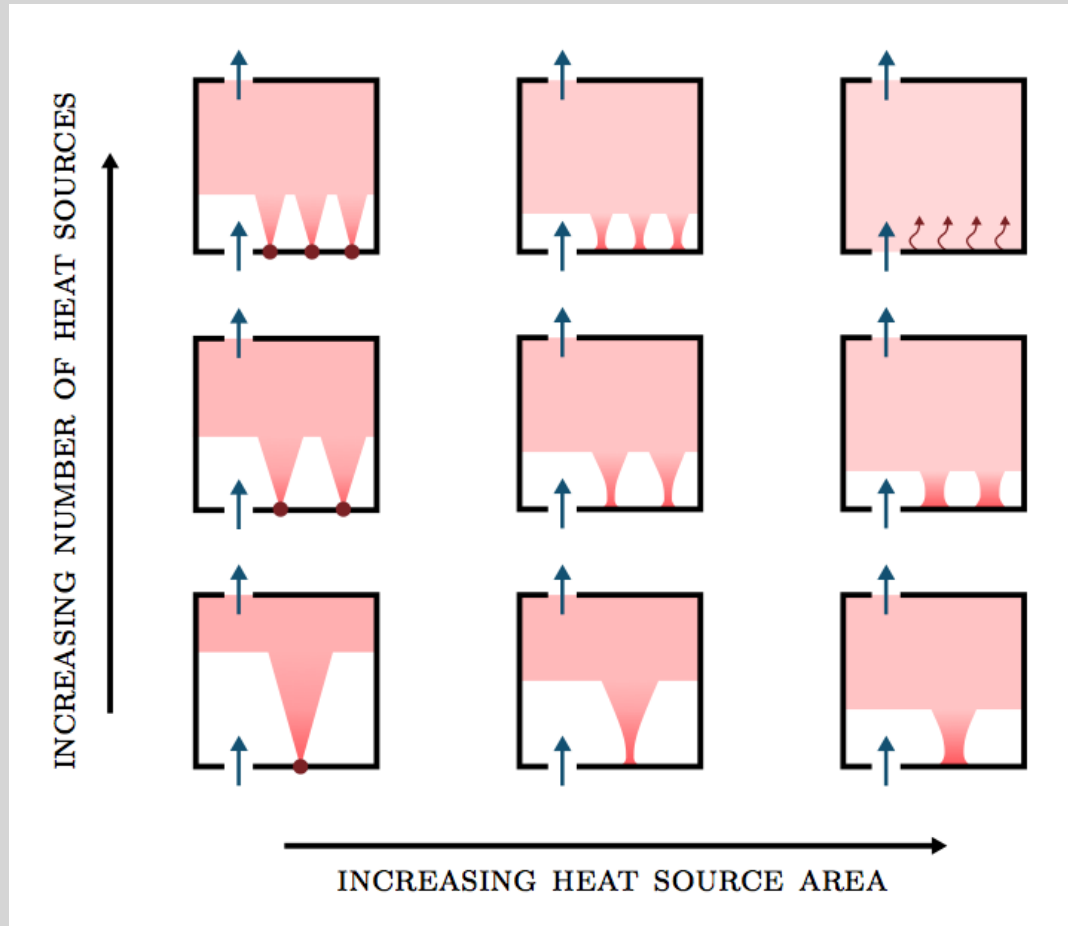
- Wires or hot plates (no volume flux)
- Smaller density differences
- Measurement by thermocouples or thermistors

## Salt in water

- Direct injection of saline
- Adiabatic walls
- Large density difference ( $\Delta\rho/\rho \sim 0.2$ ) possible
- Measurement by salinity probes or dye attenuation technique

- Dye & shadowgraph visualisation

# Modelling heat sources



‘Underfloor  
heating’

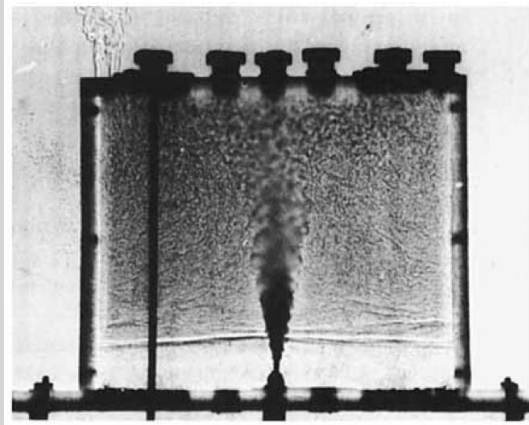
GLADSTONE & WOODS (2001)

(or ‘Chilled  
ceiling’)

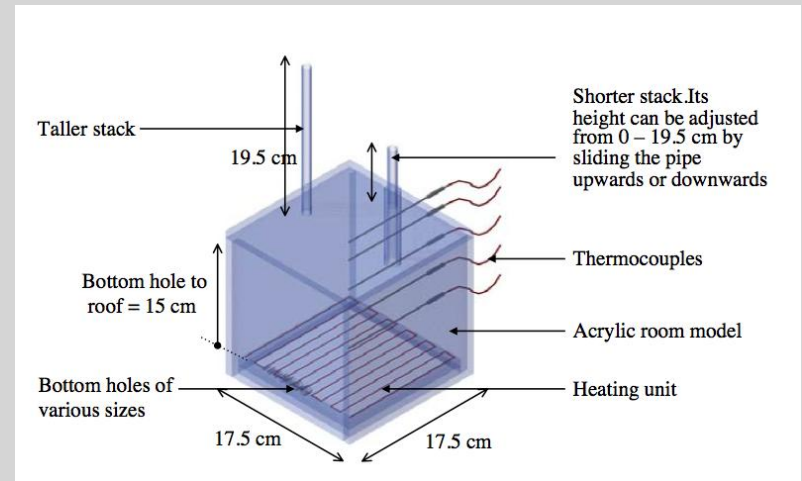
‘Single person’

LINDEN ET AL (1990)

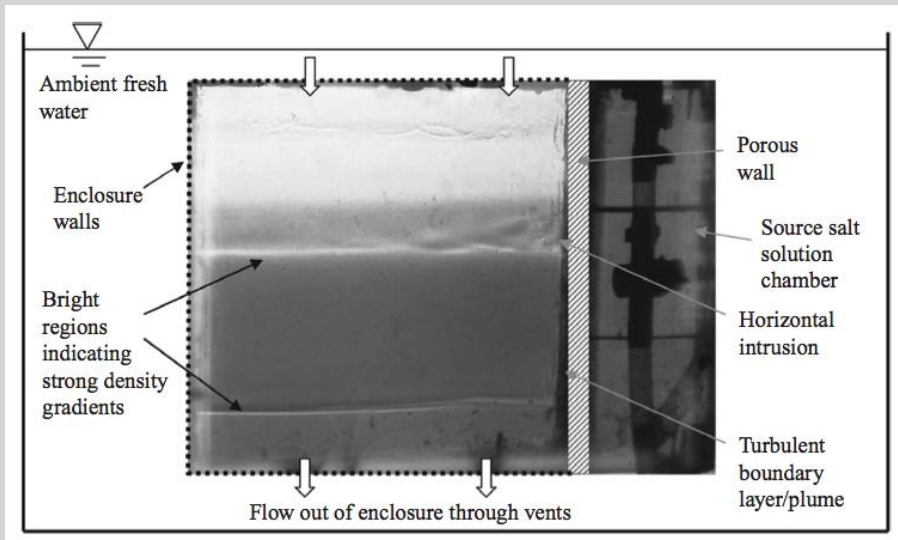
# Modelling heat sources



LINDEN ET AL (1990)



CHENVIDYAKARN & WOODS (2005)

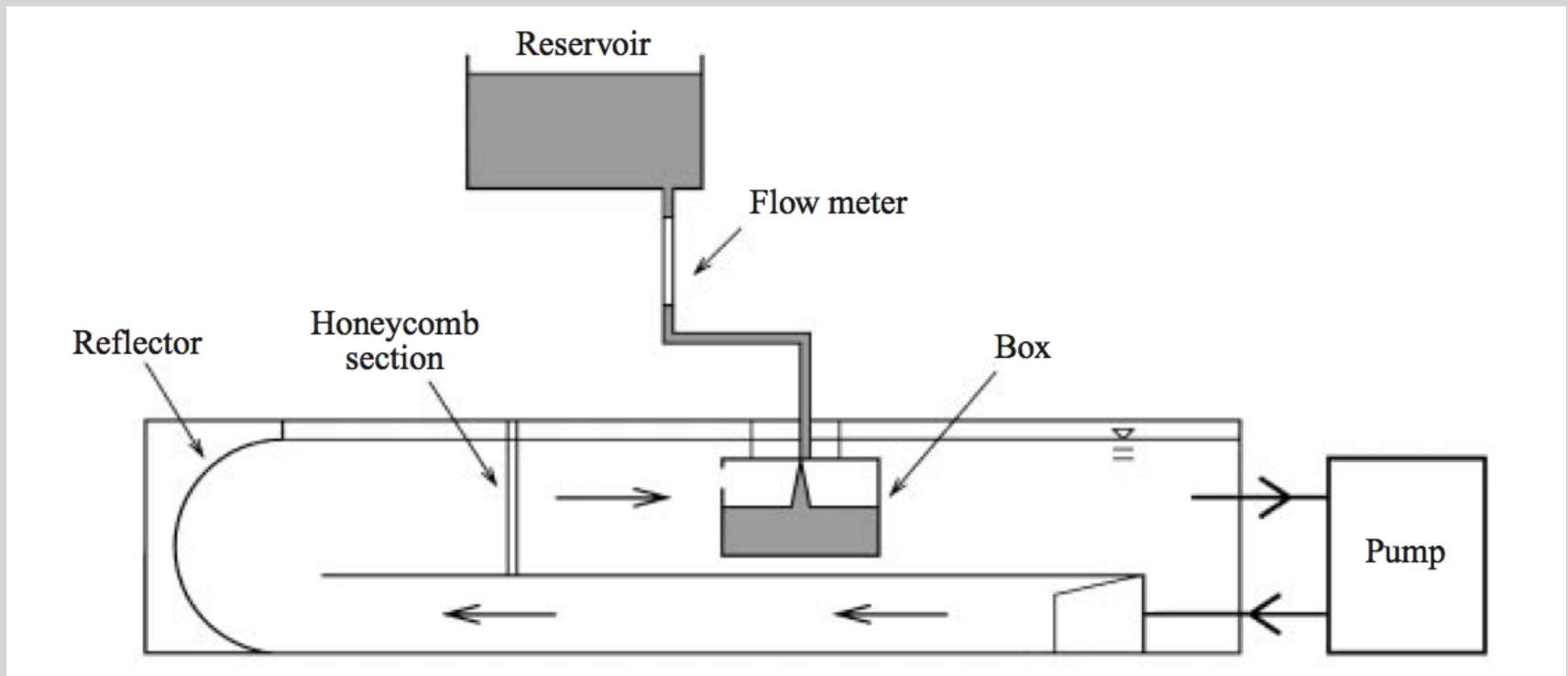


COOPER & HUNT (2010)



LIVERMORE & WOODS (2007)

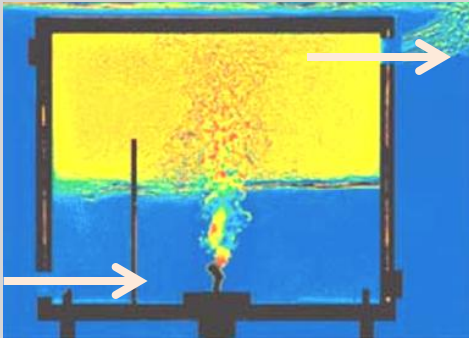
## Salt in water with a recirculating flume



HUNT & LINDEN (2001)

## Multiple steady flow regimes

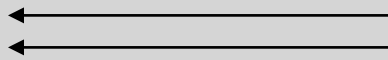
‘No wind’



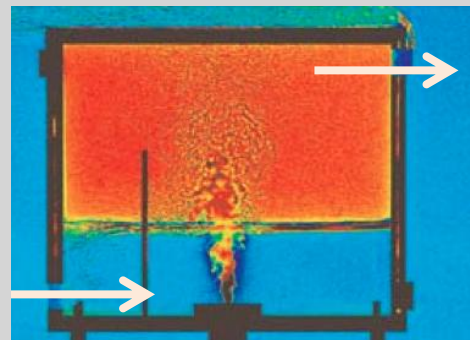
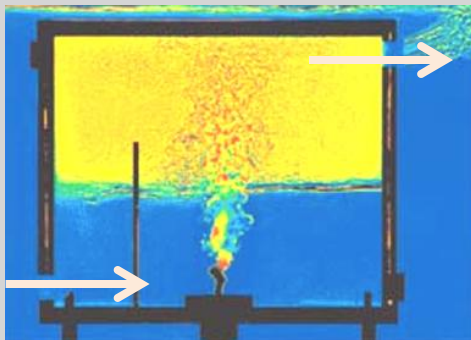


## Multiple steady flow regimes

‘Opposing wind’



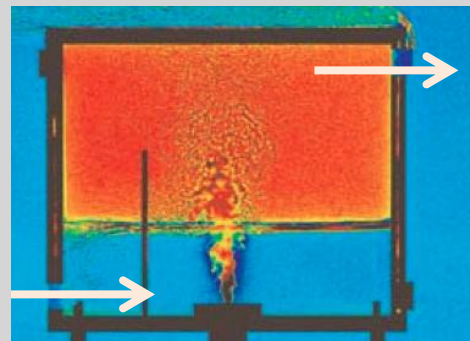
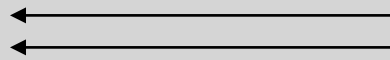
‘No wind’



Forward  
displacement flow

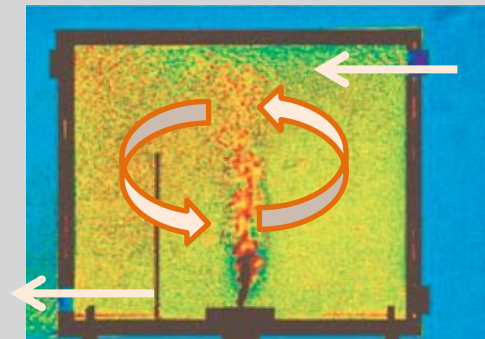
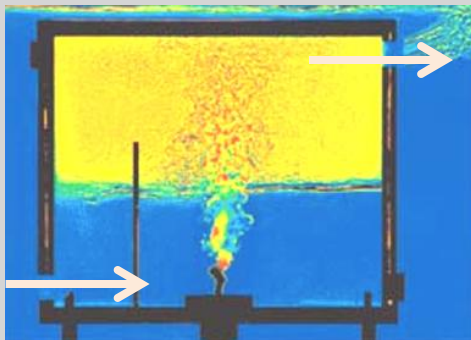
## Multiple steady flow regimes

‘Opposing wind’



Forward  
displacement flow

‘No wind’

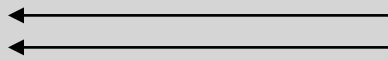


Reversed flow  
with mixing

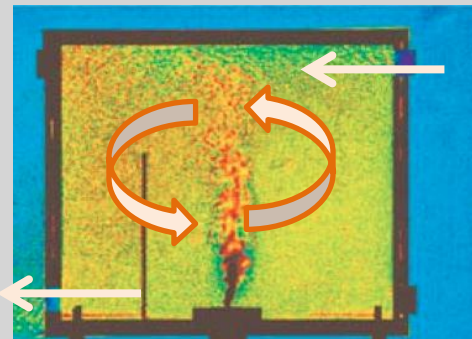
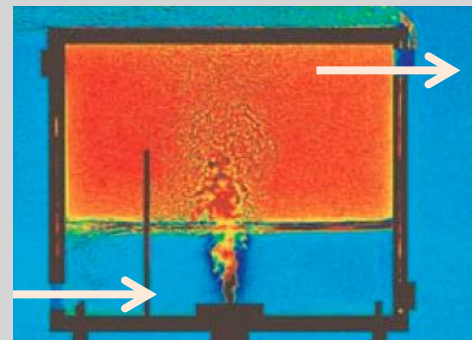
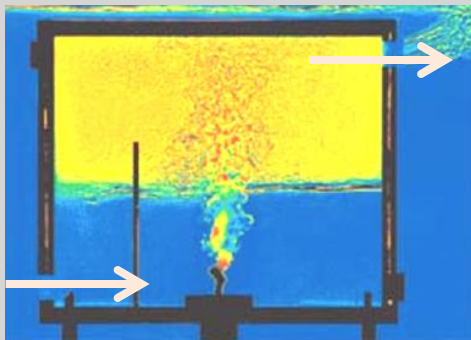
# Combining wind & buoyancy

## Multiple steady flow regimes

‘Opposing wind’



‘No wind’

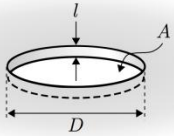


Time history  
dependent

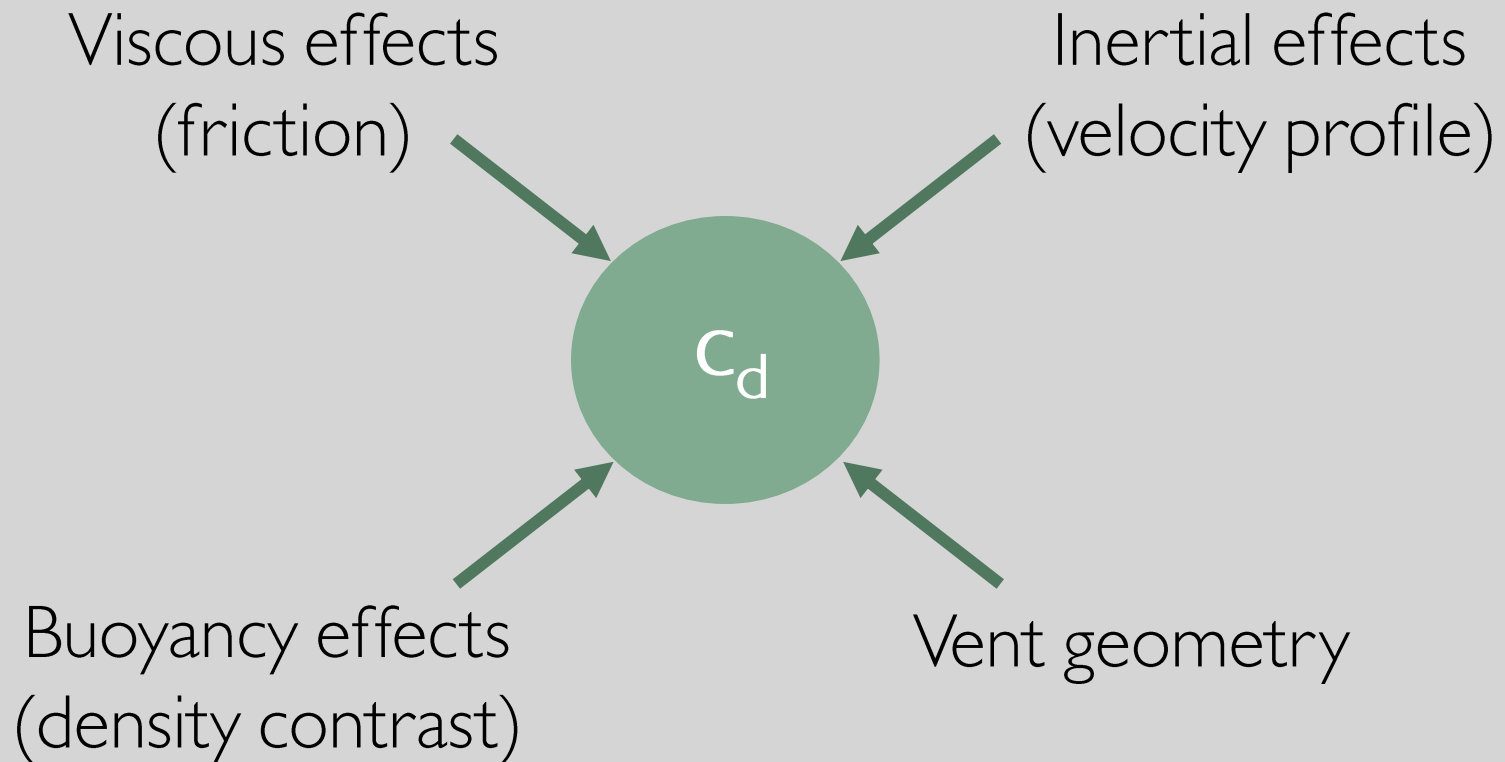
Forward  
displacement flow

Reversed flow  
with mixing

# $c_d$ and exchange flows



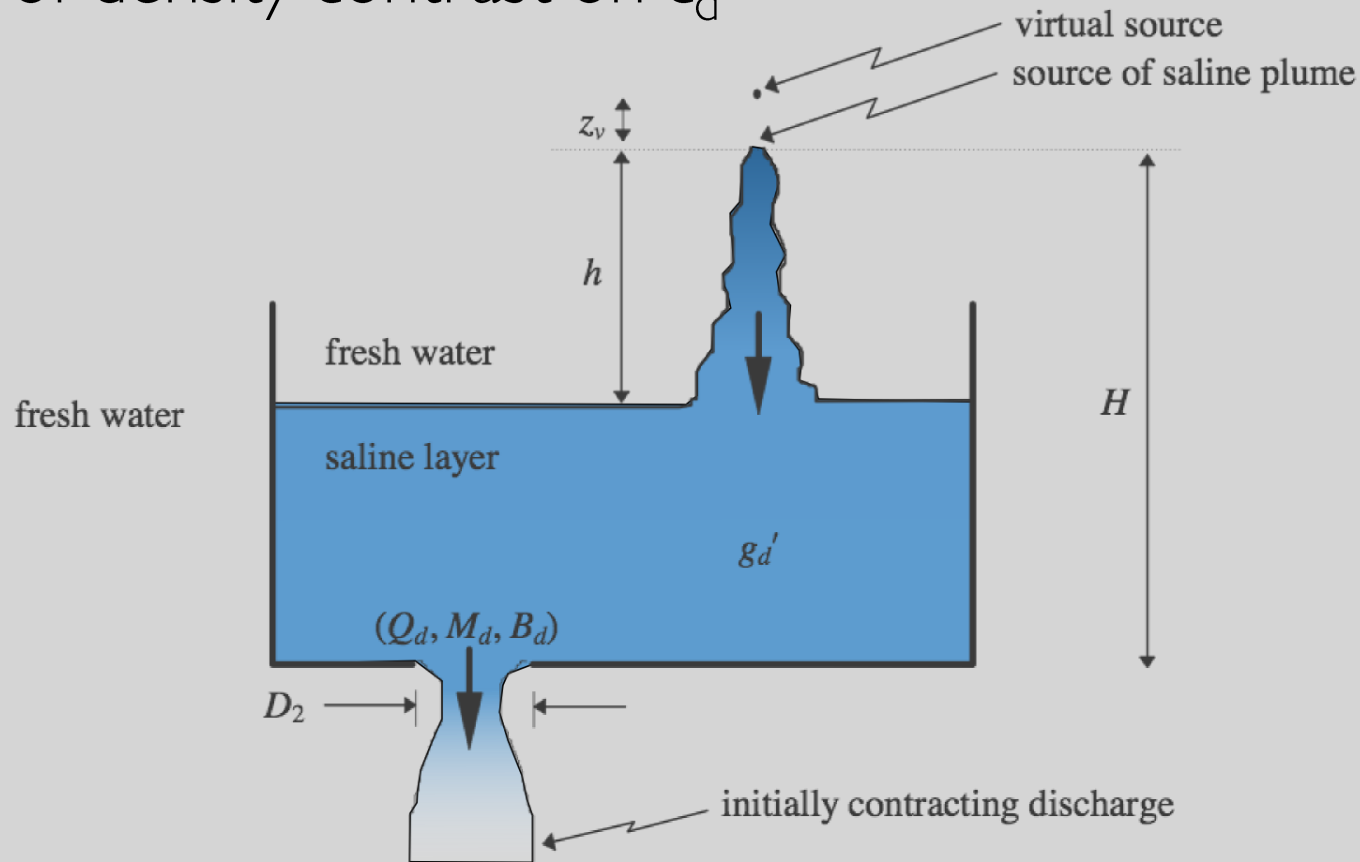
$$A_{\text{effective}} = c_d A_{\text{geometric}}$$



# $c_d$ and exchange flows



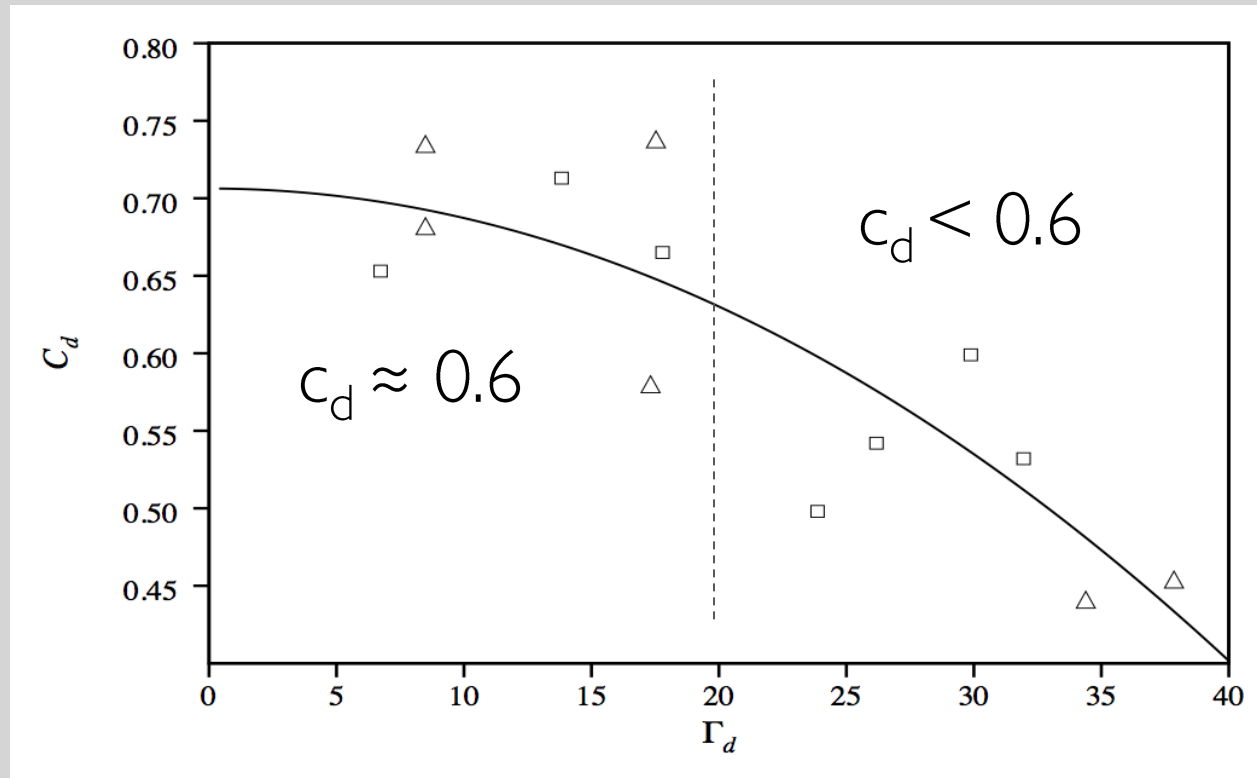
Holford & Hunt (2000 & 2001):  
Experimental investigation of effect  
of density contrast on  $c_d$



# $c_d$ and exchange flows

Increasing density contrast  
AND/OR Increasing vent size  
AND/OR Decreasing flow rate

} Decreasing Fr

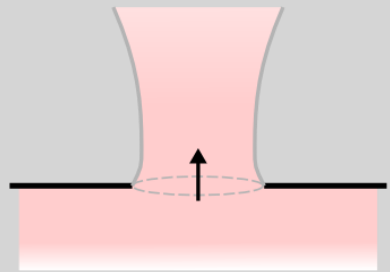
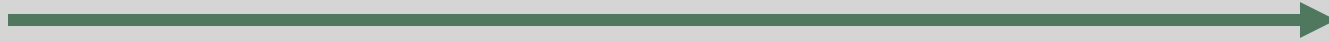


# $c_d$ and exchange flows

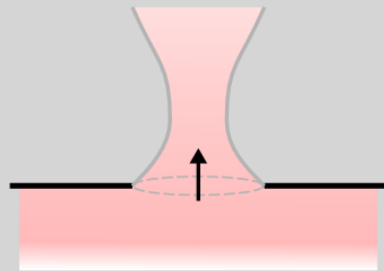


Increasing density contrast  
AND/OR Increasing vent size  
AND/OR Decreasing flow rate

} Decreasing Fr



'Normal'  
unidirectional  
flow

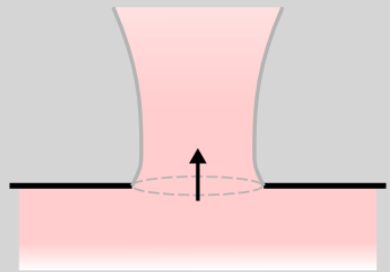


Significant  
necking,  
decrease in  $c_d$

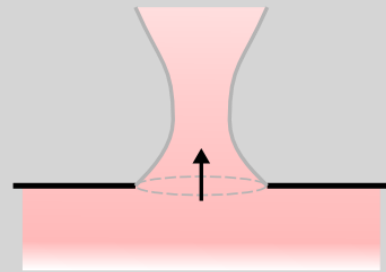
# $c_d$ and exchange flows

Increasing density contrast  
AND/OR Increasing vent size  
AND/OR Decreasing flow rate

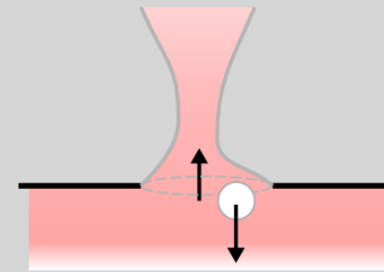
} Decreasing  $Fr$



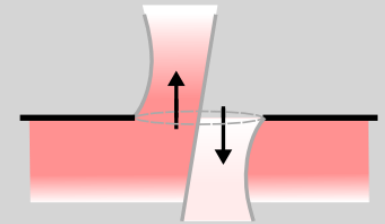
'Normal'  
unidirectional  
flow



Significant  
necking,  
decrease in  $c_d$



Pulsing



Steady

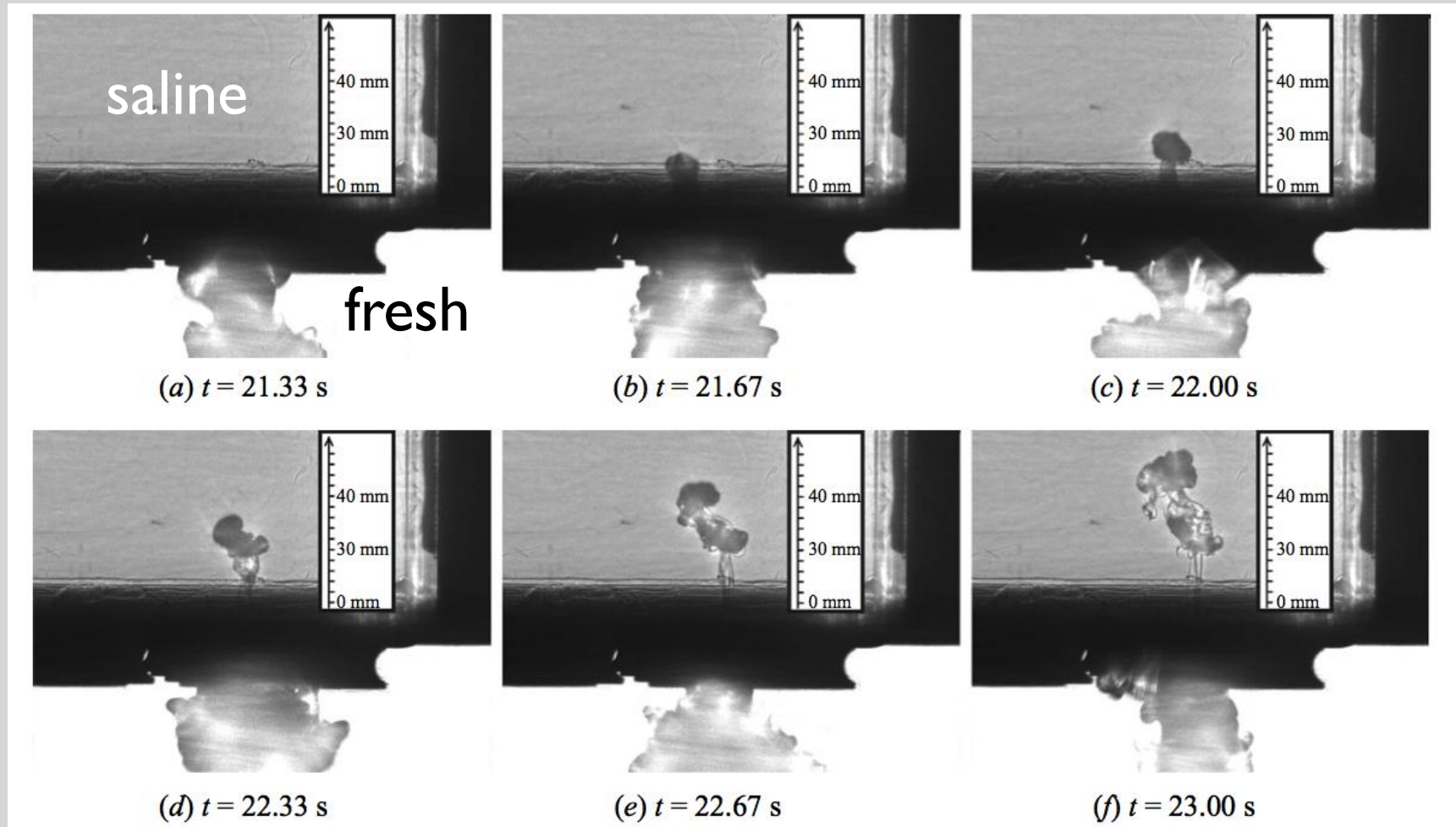


Exchange flows



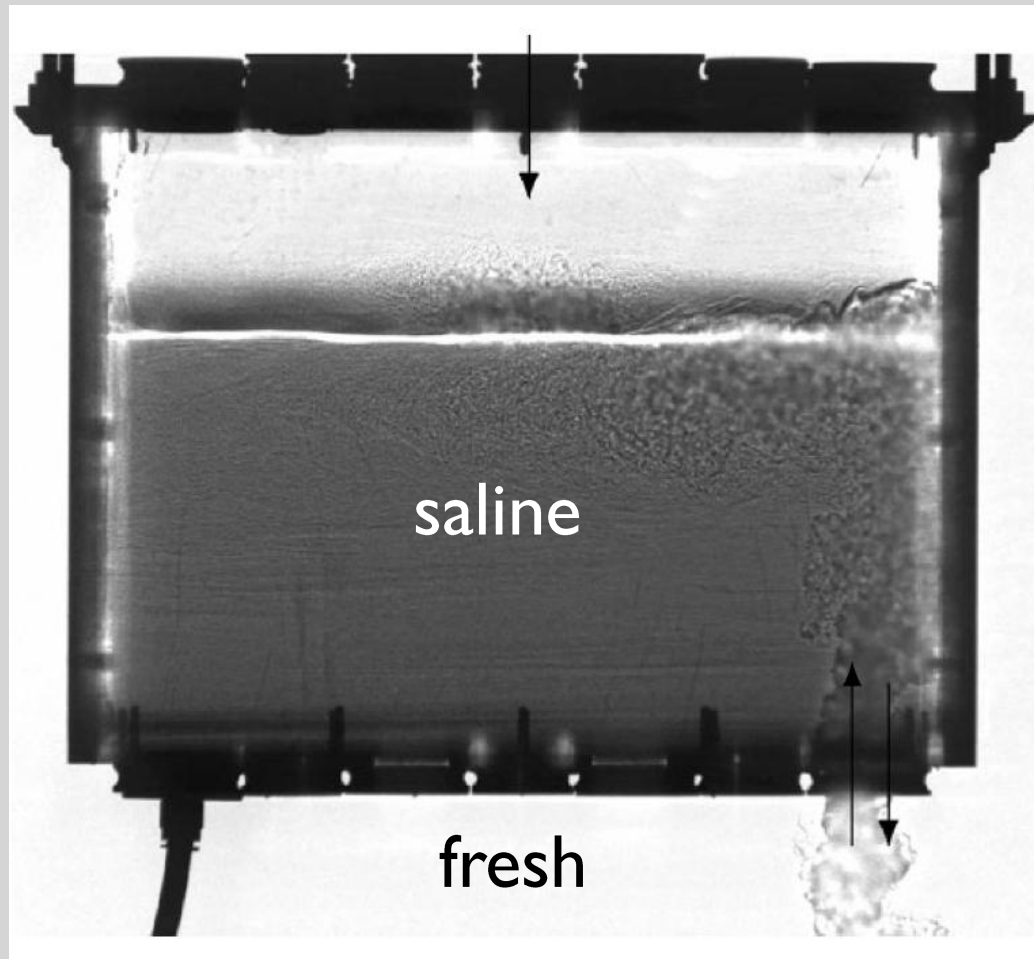
# $c_d$ and exchange flows

## Pulsing exchange flow



# $c_d$ and exchange flows

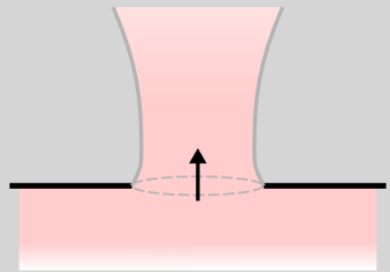
Steady exchange flow



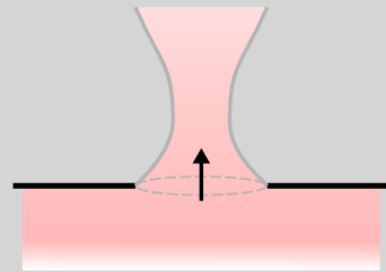
# $c_d$ and exchange flows

Increasing density contrast  
AND/OR Increasing vent size  
AND/OR Decreasing flow rate

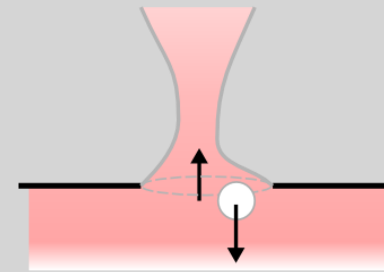
} Decreasing  $Fr$



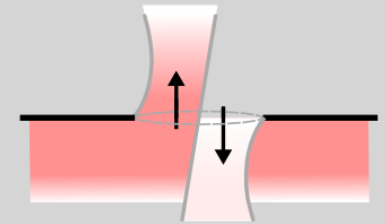
'Normal'  
unidirectional  
flow



Significant  
necking,  
decrease in  $c_d$



Pulsing



Steady



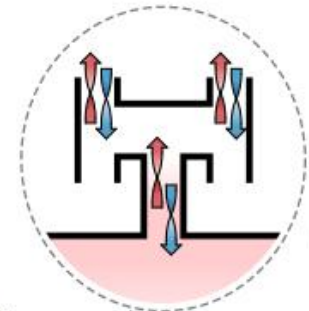
Exchange flows

## Example application: The Contact Theatre

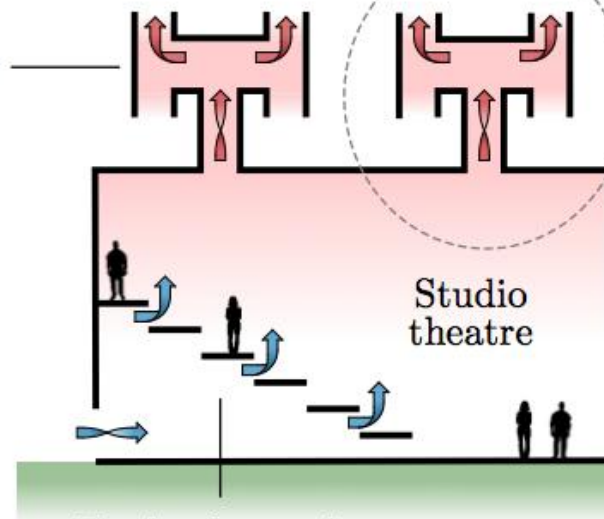


H-pot stacks

Bidirectional flow  
(also observed)



Unidirectional outflow  
(intended scheme)



Studio  
theatre

Fresh air supply

## Key advantages

- Ability to visualise real-time, 3D flows
- Ability to isolate and quantify specific flow phenomena

## Key drawbacks

- Need specialist facility
- Difficult to recreate highly dynamic 'real' building scenarios, to capture daily building usage, weather conditions etc.

**Chenvidyakarn & Woods (2005)** *Multiple steady states in stack ventilation*. Building & Environment 40, 399-410.

**Cooper & Hunt (2010)** *The ventilated filling box containing a vertically distributed source of buoyancy*. Journal of Fluid Mechanics 646, 39-59.

**Hunt & Coffey (2010)** *Emptying boxes – classifying transient natural ventilation flows*. Journal of Fluid Mechanics 646, 137-168.

**Hunt & Holford (2000)** *The discharge coefficient - experimental measurement of a dependence on density contrast*. Proceedings of the 21<sup>st</sup> AIVC Conference.

**Hunt & Linden (2001)** *Steady-state flows in an enclosure ventilated by buoyancy forces assisted by wind*. Journal of Fluid Mechanics 426, 355-386.

**Hunt & Linden (2005)** *Displacement and mixing ventilation driven by opposing wind and buoyancy*. Journal of Fluid Mechanics 527, 27-55.

**Larice (2009)** *Classifying steady states in emptying-filling boxes*. PhD Thesis, Imperial College London.

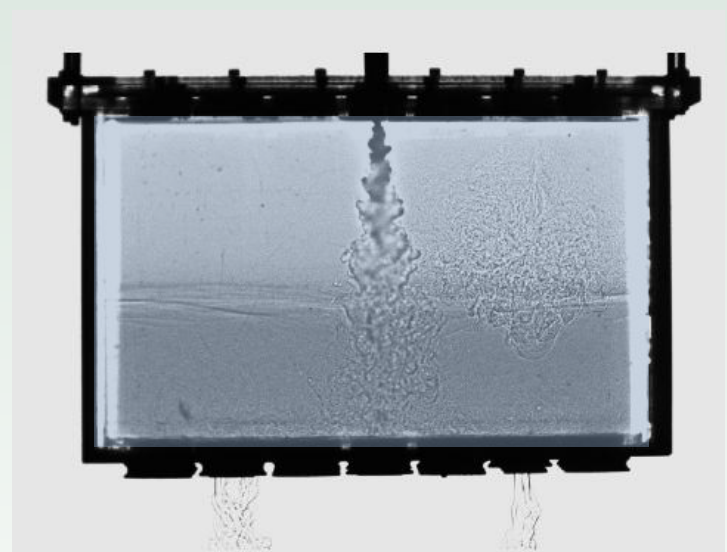
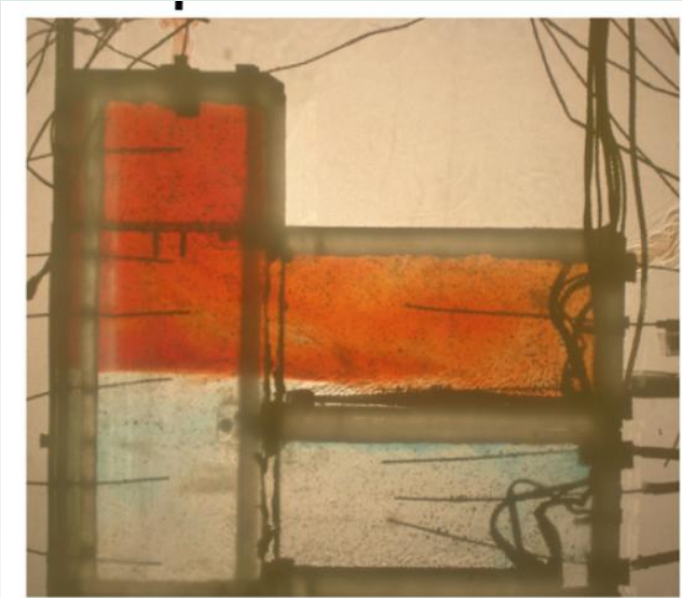
**Linden et al (1990)** *Emptying filling boxes : the fluid mechanics of natural ventilation*. Journal of Fluid Mechanics 212, 309-335.

**Livermore & Woods (2007)** *Natural ventilation of a building with heating at multiple levels*. Building & Environment 42, 1417-1430.

# Modelling natural ventilation: Small-scale experiments in water

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